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Matching Seals to Dynamic Sealing Applications

Understanding a seal's function isn't difficult—it keeps fluids in and contaminants out. Understanding how to get this done most effectively in a specific situation is an entirely different matter.

Beth Figliulo | Dec 07, 2018

Successful sealing of fluid power cylinders requires a delicate balance of interrelated factors that affect the sealing process in a dynamic system. These factors include lubrication and mating surfacUUes; design factors, such as shape, material, and process; and environmental factors, such as pressure, temperature, time, dynamics, and assembly.

Understanding the role of lubrication is the first key to successful sealing in dynamic systems. A fully lubricated system will experience some leakage, usually in the form of a fluid film, but friction will decrease, leading to a corresponding decrease in seal wear. Not surprisingly, the opposite is true: In a sparsely lubricated system, the fluid film decreases, but friction—and thus, wear—increase.

Moreover, sealing tends to improve as fluid pressures increase. Because system pressure on the seal surface compresses the seal axially, the seal is forced more tightly into the gland, improving conformability of the seal with the metal surfaces around it. If the seal is correctly designed, as system pressure increases, sealing force and effectiveness also increase.

A seal's ability to resist extrusion is a third key factor. Seals in dynamic systems must resist shear forces resulting from the differential between the pressurized and unpressurized sides of the seal. Shear forces tend to push the seal into the gap between adjacent metal surfaces (known as the clearance gap), and the seal material and profile must be sufficiently strong and stiff to resist being damaged or destroyed.

Rod Sealing

Leak-free performance and high service life cannot be ensured by a single sealing element in heavy-duty applications. Because the cylinder's piston rod is directly exposed to the environment, it's paramount that the seals keep the fluid in the system. External leakage is easily noticed. Not surprisingly, then, it has become a common criticism of hydraulic power transmission.

The solution is redundant systems that involve specially developed seals arranged in a series, often referred to as *in tandem*. Each sealing element in such systems has a specific function, and their interaction is critical to a high-functioning system.



This illustration shows a variety of piston rod seals to make an effective sealing system. At left and center are tandem wear rings, at far right is a rubber-energized double-acting scraper seal, and the other two are single-acting O-Ring-energized rod seals.

Two main configurations are recommended for tandem sealing systems. The first is a vented unidirectional polytetrafluoroethylene (PTFE) slipper seal with either a unidirectional PTFE or polyurethane slipper seal as the secondary seal. This configuration has the advantage of limiting micro fluid-film formation under the seal and ensuring that when the rod is retracted, the sealing face supports hydrodynamic back pumping of the fluid film. This results in leak-free sealing with low friction for decreased wear. Additionally, the secondary seal materials described above provide superior sliding ability with maximum wear resistance.

The second configuration is a buffer seal, which has an integrated backup ring and a polyurethane U-cup as the secondary seal. This arrangement is usually more cost-effective than the first option and may suffice for systems requiring high abrasion resistance or where pressure will continually fluctuate between high and low pressure. A modified version of the buffer seal also can be used to provide lubrication for the U-cup so it does not run dry.

Rod scrapers—Contamination of the hydraulic fluid is the most common cause of damage to seals, rings, and other components. O-ring energized scrapers and double-lipped polyurethane scrapers are recommended for dynamic sealing systems, with the choice depending on the type of contaminants expected and the friction and stick-slip requirements of the overall system.

O-ring energized scrapers are composed of PTFE or another thermoplastic material and are installed with an elastic O-ring in one groove. The O-ring maintains the pressure of the scraper lips against the sliding surface, which compensates for deflections in the piston rod and allows effective scraping of contaminants—even firmly adhered dirt.

Double-lipped polyurethane scrapers are superior to conventional elastomeric scrapers, with a scraper lip designed to effectively remove dirt while leaving the oil film required for correct operation. The sealing lip faces inward to assume a sealing function under low pressure, and the static seal is achieved by a tight radial fit between the scraper body and the groove.

Piston Sealing

Choosing a piston seal should be based on its wear and gap extrusion resistance, compatibility with the hydraulic fluid, operating temperatures, sliding ability, and ease of installation. The mating surface is also a key consideration—specified limits should be observed since they can greatly affect the piston seal's service life.

Three types of bi-directional slipper seals are recommended for dynamic piston systems: either a PTFE or polyurethane seal with an optimized cap

profile based on the material or a seal with a PTFE cap that contains an integrated elastomer element. The choice depends on the media to be sealed and the desired friction performance of the system.

• PTFE slipper seals are ideal for applications requiring low friction and no stick-slip or where larger allowable clearance gaps are present.

• PTFE slipper seals with an integrated elastomer element are ideal for situations where fluid separation or tight leakage control are required.

• Polyurethane slipper seals offer high abrasion resistance and are a good fit for situations where blow-by could be a concern.



PTFE slipper seal shown in a vented unidirectional configuration.

When redundancy against leakage is required, unidirectional slipper seals or polyurethane U-cups can be added on one or both sides (depending on pressure direction and/or fluctuation) of the bidirectional seal to further prevent leakage and ensure media separation.

Wear Rings

Wear rings absorb side-load forces occurring on the piston or piston rod of a cylinder and eliminate metal-to-metal contact between the sliding parts of the cylinder. Non-metallic wear rings offer significant benefits over traditional metal rings. Non-metallic wear rings have a longer service life, higher load-bearing capacity, lower friction, and good scraping effects. They also damp mechanical vibrations to reduce noise and are cost-effective.

In terms of material, PTFE-based wear rings are recommended for low- to medium-duty applications with limited radial forces. The low friction of PTFE rings provides smooth operation at low speeds without stick-slip motion. These rings should be selected for high wear resistance and good damping capability.

Rings composed of glass-filled nylon are recommended for applications with medium- to heavy-duty radial forces. Look for rings offering high compressive strength at high temperatures and easy installation on pistons and glands.

Fabric-reinforced composite wear rings are the best choice for heavy-duty applications with high radial forces. These rings should distribute high radial forces evenly and exhibit good sliding and dry-running properties, good scraping effect, and high wear resistance.

High-Pressure Cylinders

Corner-reinforced seals are an excellent choice for high-pressure applications (up to 15,000 psi) because they are highly resistant to extrusion. These seals contain an integrated ring made of poly ether keytone (PEEK) or polyacetal and can accept 50% to 100% higher extrusion gaps compared to those without corner reinforcement.





Corner-reinforced seals perform well in high-pressure applications.

PTFE and polyurethane seals are available with corner reinforcements for both rod and piston configurations. Some are designed for unidirectional use while others provide bidirectional sealing. In unidirectional seals, look for versions that are designed to stabilize the seal against back-pressure by preventing the seal from tipping forward in the groove.

Mating Surface Finish and Coatings

Many design engineers are aware of the importance of mating surface finish on seal functionality. The mating surface's finish can greatly impact:

- Friction and heat—The rougher the surface of the hardware, the greater the amount of friction and heat generated.
- Wear—Rougher surfaces lead to more wear, as do hard surfaces.
- Sealing ability—The rougher the surface, the less effective the sealing ability.
- Long-term cost cost—Field failures and repairs are expensive, so long-wearing, high-performing seals ultimately save money.

R_a = arithmetical mean roughness value

R_p = maximum profile peak height

 $R_{-} = mean roughness$

Ra = 2.5 µm



Average roughness does not provide enough detail to properly define surface finish. Instead, a combination of surface measurements must be considered.

The optimum sealing profile includes microscopic valleys and avoids a high concentration of peaks. Valleys are needed to build up a sufficient fluid film to reduce friction and wear; a high concentration of peaks can cause excessive seal wear. However, simply knowing the roughness average (R_a) value of the surface finish is not enough—completely different surfaces can have the same R_a value. In addition to roughness average, consider maximum profile peak height, mean roughness depth, profile bearing ratio, degree of skew, and total height of the roughness profile.

Slipper seals run well against unplated surfaces at moderate speeds and pressures, but harder surfaces are recommended in high-speed and high-pressure reciprocating applications. The softer the metal, the more likely the seal will polish its mating surface during a run-in period (the initial timeframe of high wear and friction). Conversely, harder surfaces can accelerate seal wear. Therefore, optimize the seal profile so an adequate fluid film is maintained. In addition, seal materials that have more robust fillers, such as bronze or PEEK, can be used to help withstand harder surfaces during the run-in period.





The peaks on the counter surface are broken away during the initial start-up or break-in period.

Once the peaks on the mating surface are broken off and the hardware and seal reach a state of equilibrium, the run-in period ends. Seal friction and wear decrease after the run-in period, so reduce this period as much as possible by precisely controlling the surface profile, especially on harder mating surfaces. This will ensure the system runs as efficiently as possible from the start and helps increase overall seal life.

Future Trends

One important trend to watch is the move away from chrome plating on parts used in hydraulic systems. Chrome is still the most commonly used plating in the industry, but hexavalent chromium has been identified as a carcinogen, and the process to make chrome plating produces toxic waste. As new restrictions on the chemicals being used for chrome plating and the waste being created are introduced, plating options that are more environmentally friendly are likely to gain popularity.

From a sealing perspective, it will be critical to understand how commonly used materials interact with the new plating types. Seal manufacturers will need to evaluate how friction, wear, and general sealability are affected by the different plating options and either make modifications to existing seals or develop new materials to ensure high performance and long wear in the newer systems.

Seals in Pneumatic Cylinders

Pneumatic cylinders normally operate at significantly lower pressures than hydraulic cylinders do. Also, pneumatic cylinders and valves often operate at high speeds and high cycling rates because they can exhaust air straight into the atmosphere when changing direction, rather than routing hydraulic oil through a return line.

Although many older pneumatic systems use lubricated air, most new systems use dry, unlubricated air. The minimal lubrication present in these systems can present a sealing challenge. An initial application of grease can help with the initial movement, and if the seal is able to maintain that lubrication film, the initial grease used may be able to extend seal life without the need for additional lubricant. Importantly, contaminated air will slowly compromise the lubricant, shortening the seal life.

In regard to friction, it is advisable to provide extra calculated force to the cylinder to account for overcoming internal friction from seals and bearings or even pressure losses from clogged filters or component restrictions in the circuit. There could also be pressure losses due to leaks over time. To avoid such challenges, seal profiles similar to those used in hydraulic cylinders (U-Cups and slipper seals) should be optimized for lower friction, preserving the lubrication film and minimizing the chance of leakage.

Polyurethane is a commonly used material due to its abrasion and wear resistance, both of which are critical in an oil-free environment. In situations where low friction and no stick-slip are required, PTFE and other thermoplastics can be used. To address high-pressure peaks, cylinder cushioning occurs in doubleacting cylinders where a plunger deploys to reduce the speed of the piston at the end of the stroke or hit of the end cap. There are certain seals optimized for pneumatic cushioning in situations where high-pressure peaks may occur. The seals are designed with an integrated check valve to ensure pressure application on the full piston surface once the pressure direction changes. It lessens the impact of stresses on the interior hardware, extending the cylinder's operational life and reducing potential maintenance issues.

Wear rings in modified polyoxymethylene (POM) materials are often used due to their ability to operate in dry conditions. They also have high compressive strength and good resistance to abrasion. The wear ring has a dual function of guiding the piston or the rod in a cylinder and absorbing the transversal forces within it.

SURFACE ROUGHNESS, µM		
Parameter	Mating surface	Groove surface
R _{max}	1.0 to 4.0	< 16.0
R _{z DIN}	0.63 to 2.5	< 10.0
Ra	0.10 to 0.60	< 1.6

Pneumatic cylinders typically have higher surface finish requirements, due to the ability of gas to permeate potential imperfections.

In any of the scenarios discussed, the seal profile must be carefully assessed for proper selection. The primary factors affecting seal selection include temperature, wear resistance, abrasion, sealed pressure, seal materials and their associated attributes, compatibility, vibration, and expected life.

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